

# The Stability of Frictional Sliding on Dip-Slip and Finite Length Faults

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## 1 Abstract

This paper examines the linear stability of sliding on faults embedded in a 2D elastic medium that obey rate and state friction and have a finite length and/or are near a traction-free surface. Previous work typically has examined either spring-slider systems or infinitely long faults. Results are obtained using a numerical technique that allows for analysis of systems with geometrical complexity, and/or heterogeneous material properties; however only systems with homogeneous frictional and material properties are examined. Some analytical results are also obtained for the special case of a fault that is parallel to a traction-free surface. On velocity-weakening faults with finite length, there is a critical fault length  $L^*$  for unstable sliding that is analogous to the critical nucleation length  $h^*$  that is usually derived from spring-slider or infinite fault systems. Faults longer than  $L^*$  are unstable to perturbations of any size. On vertical strike-slip faults or faults in a full-space  $L^* \approx h^*/e$ , where  $e$  is Euler's number. For dip-slip faults near a traction-free surface  $L^* \leq h^*/e$  and is a function of dip angle  $\beta$ , burial depth  $d$ , and friction coefficient. In particular,  $L^*$  is at least an order of magnitude smaller than  $h^*$  on shallowly dipping ( $\beta < 10^\circ$ ) faults that intersect the traction-free surface. Additionally,  $L^* \approx h^*/e$  on dip-slip faults with burial depths  $d \geq h^*$ . For sliding systems that can be treated as a thin layer, such as landslides, glaciers, or ice streams,  $L^* = h^*/2$ . Finally, conditions are established for unstable sliding on infinitely-long, velocity-strengthening faults that are parallel to a traction-free surface.

**Key words:** Friction, Instability analysis, Earthquake dynamics.

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## 2 Introduction

How does the geometry of fault systems affect the stability of frictional sliding? Most knowledge of frictional stability comes from analyses of spring-slider systems or systems consisting of an infinitely long fault in an infinite elastic medium (e.g. *Ruina, 1983; Rice and Ruina, 1983; Rice et al., 2001; Uenishi and Rice, 2003*). These two systems do not include important aspects of fault geometry such as proximity to a traction-free surface, fault dip angle, or finite fault length. This paper examines the linear stability of sliding on finite length faults that obey rate and state friction (RSF) and are embedded in a 2D elastic continuum with homogeneous material properties. As in many previous studies, the focus here is on the quasi-static regime where inertia is neglected (e.g. *Rice and Ruina, 1983; Uenishi and Rice, 2003; Viesca, 2016a,b; Aldam et al., 2017; Heimisson et al., 2019; Ozawa et al., 2024*).

The specific finite length geometries considered are: faults in an infinite full-space; faults parallel to a traction-free surface; and dip-slip faults and vertical strike-slip faults in a half-space. Including these features provides more accurate assessments of the sliding stability of natural fault systems. First, because all faults have a finite length and second, because many faults are near the surface of the earth or the seafloor. Additionally, landslides (*Handwerker et al., 2016*), ice streams (*Lipovsky and Dunham, 2017*), and glaciers (*Zoet et al., 2020*) also exhibit sliding behavior that can be described by frictional processes. The results show that these basic geometrical effects cause significant departures from long-standing results on stability behavior.

For a fault in an elastic medium, the stability of sliding can be assessed by considering a balance between the rate at which elastic stress stored in the fault system can be unloaded, and the rate at which shear stress on the fault is reduced (i.e. fault weakening) in response to sliding (e.g. *Scholz, 2019*). Unstable sliding initiates when the fault weakening rate is higher than the elastic unloading rate. In addition to earthquakes, slow frictional slip events (such as that occur in subduction zones and for example on the Whillians Ice Plain) are also a manifestation of unstable behavior since they involve episodic slip that is faster than the long term slip rate (e.g. *Bindschadler et al., 2003; Schwartz and Rokosky, 2007; Bürgmann, 2018*).

For frictional sliding, the changes in shear stress on a fault are described by the RSF equations wherein the evolution of the friction coefficient  $\mu$  on a surface is a function of the sliding rate  $v$  and

53 an internal state variable  $\theta$

$$\mu(v, \theta) = \mu_0 + a \ln\left(\frac{v}{v_0}\right) + b \ln\left(\frac{v_0 \theta}{d_c}\right), \quad (1)$$

54 where  $\mu_0$  is a reference friction coefficient for steady state sliding at the reference velocity  $v_0$ ,  $a$   
 55 and  $b$  are constitutive parameters, and  $d_c$  is a critical slip distance that is related to the amount of  
 56 slip needed to attain a new steady state after changes in sliding velocity (*Dieterich, 1979; Ruina,*  
 57 *1983; Marone, 1998*). The reference values  $\mu_0$  and  $v_0$  are not arbitrary, but are determined from  
 58 experimental measurements of how the steady state friction coefficient of a given material depends  
 59 on the sliding velocity. However, pairs of values  $(\mu_0, v_0)$  can be chosen from amongst a set of such  
 60 measurements. The evolution of the state variable is commonly described using the aging or slip  
 61 laws:

$$\text{aging law: } \frac{\partial \theta}{\partial t} = 1 - \frac{v\theta}{d_c}, \quad \text{slip law: } \frac{\partial \theta}{\partial t} = -\frac{v\theta}{d_c} \ln\left(\frac{v\theta}{d_c}\right). \quad (2)$$

62 For velocity-weakening friction ( $a - b < 0$ ; or  $a/b < 1$ ), fault weakening will occur due to a  
 63 reduction in the friction coefficient as sliding rate increases. Weakening can also occur due to a  
 64 coupling between fault slip and changes in normal stress, which can lead to unstable behavior for  
 65 velocity-strengthening friction below some critical value of  $a/b$ . This effect has been shown to exist  
 66 on bimaterial and poroelastic faults (*Rice et al., 2001; Ranjith, 2014; Heimisson et al., 2019*), on  
 67 faults with fault-valve behavior (*Ozawa et al., 2024*), and on faults that lack geometric reflection  
 68 symmetry across the sliding surface (*Aldam et al., 2016*). Lack of geometric reflection symmetry  
 69 is a very general feature of fault systems. So too then is the possibility of unstable behavior on  
 70 velocity-strengthening faults. The results in this paper take a first step in establishing the range of  
 71 parameters where this behavior occurs on faults that are near a traction-free surface.

72 Linear stability analysis of faults in an elastic continuum leads to the concept of a nucleation  
 73 length  $h^*$  (*Ruina, 1983; Rice and Ruina, 1983*). The nucleation length is usually interpreted as the  
 74 minimum length of a failing fault patch that is required for an unstable sliding event to develop (e.g.  
 75 *Dieterich, 1992; Scholz, 2019*). In this interpretation, failure on velocity-weakening fault patches  
 76 with lengths smaller than  $h^*$  cannot develop into an unstable event. The value of  $h^*$  is usually  
 77 derived using one of two different methods.

78 First,  $h^*$  can be found analytically for the special case of an infinitely long fault with constant  
 79 frictional properties and effective normal stress  $\sigma$ , embedded in an infinite, 2D elastic full-space  
 80 with homogeneous properties. Allowing the fault to be infinitely long simplifies the mathematical  
 81 analysis sufficiently to obtain an equation for  $h^*$ :

$$h_F^* = \frac{\pi G' d_c}{\sigma(b-a)}, \quad (3)$$

82 where  $G'$  is the effective shear modulus (*Rice and Ruina, 1983; Rice et al., 2001*). Here the symbol  $h_F^*$   
 83 is used to denote the special value of the nucleation length for a homogeneous fault in a full-space.  
 84 Because the derivation of  $h_F^*$  involves an infinitely long fault, its proper definition is the critical  
 85 wavelength of an infinitely long perturbation to the slip velocity; perturbations with wavelengths  
 86 smaller than  $h_F^*$  will not develop into unstable events (*Rice and Ruina, 1983; Rice et al., 2001*).  
 87 Therefore, in this analysis  $h_F^*$  does not represent a minimum contact or patch length.

88 Equation (3) without the factor of  $\pi$  can also be obtained by equating the critical stiffness of  
 89 an RSF spring-slider system to the stiffness of a crack subjected to anti-plane strain conditions and  
 90 a constant stress drop; other numerical prefactors are obtained depending on the assumed stress  
 91 and strain conditions (see Table 1 in *Dieterich, 1992*). Assuming a single value of stiffness for a  
 92 fault simplifies the mathematics and allows an equation for  $h_F^*$  to be obtained from a spring-slider  
 93 analysis. However, the stiffness of a fault in an elastic continuum is a quantity that evolves through  
 94 space and time as the fault slips (e.g. *Rice and Ruina, 1983; Horowitz and Ruina, 1989*).

95 When faults are not infinitely long or when a spring-slider model cannot capture important  
 96 features of a fault system, then it becomes difficult to apply analytical methods of linear stability  
 97 analysis. In this paper these difficulties are overcome by using a numerical method for conducting  
 98 linear stability analysis of 2D finite length fault systems. The method can be applied to any  
 99 fault system for which stress change functions are available (defined in the next section), and can  
 100 accommodate features such as heterogeneous material properties or multiple faults. Analytical  
 101 results are also obtained for the special case of a fault that is parallel to a traction-free surface.

102 The results of this paper show that a perturbation of any size will nucleate an unstable sliding  
 103 event on a finite length fault once the fault length is larger than some critical value  $L^*$ . Throughout  
 104 this paper, values of  $h^*$  are referred to as “critical wavelengths” and the symbol  $L^*$  is used to

105 denote a “critical fault length”. Subscripts are used for both  $h^*$  and  $L^*$  to differentiate between  
 106 specific geometries. On vertical strike-slip faults or faults in a full-space,  $L_F^* \approx h_F^*/e$  where  $e$  is  
 107 Euler’s number. For dip-slip faults near a traction-free surface,  $L_D^* \leq h_F^*/e$  and is a function of  
 108 dip angle, burial depth of the fault’s up-dip edge, and friction coefficient. However,  $L_D^* \approx h_F^*/e$  for  
 109 dip-slip faults where the up-dip edge of the fault is buried at a depth greater than or equal to  $h_F^*$ .  
 110 The results also establish conditions for linear instability under velocity-strengthening friction on  
 111 infinitely long faults that are parallel to a traction-free surface. Finally, since the focus of this paper  
 112 is on linear stability, behavior in the nonlinear regime (e.g. rupture localization or propagation) is  
 113 not considered or examined.

### 114 3 Methods

#### 115 3.1 Linear Stability Analysis

116 Consider a fault of length  $L$  that obeys equation (1) and either of equations (2), and denote  
 117 the position along the fault by  $\xi$ . Assume also that the fault is embedded in a 2D homogeneous  
 118 elastic medium with shear modulus  $G$  and Poisson ratio  $\nu$  and define an effective shear modulus  $G'$   
 119 such that  $G' = G$  for anti-plane sliding, and  $G' = G/(1 - \nu)$  for in-plane sliding. A linear stability  
 120 analysis of the fault’s sliding motion can be conducted according to the following steps. **(1)** Write  
 121 the system of nonlinear equations governing the evolution of sliding velocity  $v(\xi, t)$  and state variable  
 122  $\theta(\xi, t)$  along the fault. **(2)** Determine a uniform steady state of the system such that  $v(\xi, t) = v_0$   
 123 and  $\theta(\xi, t) = \theta_0$ . **(3)** Obtain a linearized system of equations by computing the Jacobian matrix  $\mathbf{J}$   
 124 of the nonlinear system and evaluating it at the uniform steady state so that  $\mathbf{J}_0 = \mathbf{J}(v_0, \theta_0)$ . **(4)**  
 125 Determine the stability of the linear system by examining the eigenvalues of  $\mathbf{J}_0$ . If any eigenvalue  
 126 has a positive real part then the system is unstable.

127 **Step 1.** For quasi-static sliding, the velocity of the fault is governed by a balance between  
 128 frictional resistance  $\tau_F = \mu\sigma$  and the shear stresses resolved upon the fault  $\tau = \tau_0 + \tau_E$ , where  
 129  $\tau_E$  is the change in shear stress due to gradients in slip along the fault, and  $\tau_0$  is the shear stress  
 130 on the fault in the absence of any slip. As with the shear stress, the normal stress on the fault is  
 131  $\sigma = \sigma_0 + \sigma_E$ . The stress balance changes in time as  $\dot{\mu}\sigma = \dot{\tau}_E - \mu\dot{\sigma}_E$ , and by making use of equation

132 (1), the sliding velocity of the fault can be written as

$$\dot{v}(\xi, t) = F(v, \theta) = \frac{v}{a} \left[ \frac{\dot{\tau}_E - \mu \dot{\sigma}_E}{\sigma} - \frac{b\dot{\theta}}{\theta} \right]. \quad (4)$$

133 The evolution of the state variable can simply be written as  $\dot{\theta}(\xi, t) = H(v, \theta)$ , since only the  
 134 aging and slip laws are considered here and both state variable laws have the same linearization  
 135 (e.g. *Ruina*, 1983). The nonlinear governing equations for  $\dot{v}(\xi, t)$  and  $\dot{\theta}(\xi, t)$  are now represented by  
 136 the functions  $F(v, \theta)$  and  $H(v, \theta)$ .

137 For quasi-static sliding, the changes in shear and normal stress are functions of the slip distribu-  
 138 tion  $\delta(\xi, t)$ , so that  $\tau_E = T(\xi, \delta)$  and  $\sigma_E = N(\xi, \delta)$ . The functions  $T(\xi, \delta)$  and  $N(\xi, \delta)$  are the stress  
 139 change functions mentioned in the Introduction. These functions must be determined by solving  
 140 the appropriate 2D elasticity problem for a given fault geometry (see Appendix C for example) and  
 141 contain all necessary information about the elastic response of the system. These functions also  
 142 have the property that  $\dot{\tau}_E = T(\xi, v)$  and  $\dot{\sigma}_E = N(\xi, v)$  (e.g. *Viesca*, 2016a,b). Both  $T(\xi, \delta)$  and  
 143  $N(\xi, \delta)$  are equal to zero if there is no slip gradient.

144 **Step 2.** The uniform steady state of the system satisfies the conditions  $F(v_0, \theta_0) = 0$  and  
 145  $H(v_0, \theta_0) = 0$ . Assume that the entire fault is sliding at steady state with velocity  $v_0$ , such that  
 146  $T(\xi, v_0) = N(\xi, v_0) = 0$ . For both the aging and slip laws,  $H(v_0, \theta_0) = 0$  when  $\theta_0 = d_c/v_0$ . These  
 147 conditions satisfy  $F(v_0, \theta_0) = 0$ , so the uniform steady state of the nonlinear system is  $(v_0, d_c/v_0)$ .

148 **Step 3.** To linearize the equations about the uniform steady state, first define

$$\mathbf{w}(\xi, t) = \begin{bmatrix} v(\xi, t) - v_0 \\ \theta(\xi, t) - \theta_0 \end{bmatrix} \quad (5)$$

149 where  $\mathbf{w}(\xi, t)$  is a small perturbation away from  $(v_0, \theta_0)$ . Now the linearized equations can be written  
 150 as  $\dot{\mathbf{w}} = \mathbf{J}_0 \mathbf{w}$ . The Jacobian matrix  $\mathbf{J}_0$  is most conveniently expressed in terms of the dimensionless  
 151 variables:  $\hat{t} = (v_0/d_c)t$ ,  $\hat{v} = v/v_0$ , and  $\hat{\theta} = (v_0/d_c)\theta$ , such that the uniform steady state becomes

152  $(\hat{v}_0, \hat{\theta}_0) = (1, 1)$ . Then  $\mathbf{J}_0$  can be written as

$$\mathbf{J}_0 = \begin{bmatrix} \frac{\partial \hat{F}_0}{\partial \hat{v}} & \frac{\partial \hat{F}_0}{\partial \hat{\theta}} \\ \frac{\partial \hat{H}_0}{\partial \hat{v}} & \frac{\partial \hat{H}_0}{\partial \hat{\theta}} \end{bmatrix} = \begin{bmatrix} \left(\frac{b}{a}\right) \left[\frac{1}{b}(\hat{T}_{\hat{v}} - \mu_0 \hat{N}_{\hat{v}}) + 1\right] & \left(\frac{b}{a}\right) \mathbf{I} \\ & -\mathbf{I} \end{bmatrix} \quad (6)$$

153 where  $\hat{F}_0 = \hat{F}(\hat{v}_0, \hat{\theta}_0)$ ,  $\hat{H}_0 = \hat{H}(\hat{v}_0, \hat{\theta}_0)$ ,  $\mathbf{I}$  is the identity matrix, and  $\hat{T}_{\hat{v}}$  and  $\hat{N}_{\hat{v}}$  denote derivatives  
 154 with respect to  $\hat{v}$ . Some additional mathematical steps are provided in Appendix A. The dimensions  
 155 of  $\mathbf{J}_0$  will depend on whether the Jacobian is treated analytically or numerically.

156 **Step 4.** The eigenvalues and eigenvectors of  $\mathbf{J}_0$  determine solutions to the linearized system  
 157 ( $\dot{\mathbf{w}} = \mathbf{J}_0 \mathbf{w}$ ) of the form  $\mathbf{w}(\xi, t) \propto \mathbf{w}(k\xi)e^{pt}$ . The eigenvectors  $\mathbf{w}(k\xi)$  represent small spatial per-  
 158 turbations of wavenumber  $k$  to the uniform steady state. The eigenvalues  $p$  are the corresponding  
 159 growth rates of those perturbations. If all of the eigenvalues of  $\mathbf{J}_0$  have a negative real part then  
 160 the system is linearly stable. If any eigenvalue has a positive real part then the system is linearly  
 161 unstable (e.g. *Strogatz*, 2018).

### 162 3.1.1 Analytical Stability Analysis

163 For analytical results,  $\mathbf{I} \rightarrow 1$  in equation (6) and  $\mathbf{J}_0$  can be treated as a  $2 \times 2$  matrix. Then the  
 164 eigenvalues are found by solving the characteristic equation of  $\mathbf{J}_0$ , such that

$$p^2 + \left[1 - \frac{b}{a}(\Gamma + 1)\right] p - \left(\frac{b}{a}\right) \Gamma = 0, \quad \Gamma = \frac{1}{b}(\hat{T}_{\hat{v}} - \mu_0 \hat{N}_{\hat{v}}). \quad (7)$$

165 The eigenvalues do not need to be explicitly determined in cases where  $T_v$  and  $N_v$  are purely real  
 166 functions. Instead, the stability of the system can be determined from conditions on  $\det(\mathbf{J}_0) =$   
 167  $-(b/a)\Gamma$  and  $\text{Tr}(\mathbf{J}_0) = (b/a)(\Gamma + 1) - 1$ , where  $\det()$  and  $\text{Tr}()$  denote the determinant and trace of  
 168 the matrix, respectively. The system is unstable if either  $\text{Tr}(\mathbf{J}_0) > 0$ , or  $\det(\mathbf{J}_0) < 0$  (e.g. *Strogatz*,  
 169 2018, Figure 5.2.8); however, for all of the cases examined in this paper  $\det(\mathbf{J}_0) > 0$ . The trace  
 170 instability condition can be written as

$$\left(\frac{b}{a}\right) \left[\frac{1}{b}(\hat{T}_{\hat{v}} - \mu_0 \hat{N}_{\hat{v}}) + 1\right] - 1 > 0. \quad (8)$$

171 Equation (8) can be used to obtain analytical stability results. For example, in a spring-slider  
 172 system  $N_v = 0$  and  $T_v = -K$ , where  $K$  is the spring stiffness. Then solving equation (8) for  $K$  will  
 173 yield the usual relation for the critical spring stiffness (see Appendix B.1).

174 For faults in a 2D medium, analytical results can be obtained by specifying the functional form  
 175 of the spatial perturbation  $\mathbf{w}(k\xi)$ . For infinitely long faults there are no restrictions on the values  
 176 of  $k$  because the fault has no boundaries. Then the general solution to the linear equations is

$$\mathbf{w}(\xi, t) = \int_{-\infty}^{\infty} A(k) \exp(pt + ik\xi) dk, \quad (9)$$

177 where  $A(k)$  is determined by a Fourier transform of the initial conditions (*Pivato, 2010*). The steps  
 178 of deriving equation (3) using equations (8) and (9) are detailed in Appendix B.2.

179 For a finite length fault with a uniform steady state velocity  $v_0$ , the sliding velocity must remain  
 180  $v_0$  at the boundaries and so  $\mathbf{w}(k\xi)$  must be equal to zero at the boundaries. If the fault is defined  
 181 over  $\xi = [0, L]$ , the general solution to the linear equations that satisfies these boundary conditions  
 182 is

$$\mathbf{w}(\xi, t) = \sum_n \mathbf{A}_n e^{pt} \sin(n\pi\xi/L), \quad (10)$$

183 where the constants  $\mathbf{A}_n$  are determined by Fourier series expansion of the initial conditions (*Pivato,*  
 184 *2010*). The allowable wavenumbers are  $k = n\pi/L$  (for  $n = 1, 2, \dots$ ), analogous to the normal modes  
 185 on a vibrating string. The possible values of  $k$  are discrete on a finite length fault and scaled by the  
 186 fault length  $L$ , and this has important consequences for the stability behavior.

### 187 3.1.2 Numerical Stability Analysis

188 To conduct a linear stability analysis numerically, the fault can be discretized into  $n_e$  elements  
 189 of length  $d\xi$ . Then  $\mathbf{J}_0$  becomes a  $2n_e \times 2n_e$  block matrix where in general the upper-left block  
 190  $\partial\hat{F}_0/\partial\hat{v}$  is dense and the other blocks are sparse diagonal matrices. In discrete form, the functions  
 191  $T(\xi, v)$  and  $N(\xi, v)$  become linear operators on the slip velocity. For example,  $T(\xi, v) \rightarrow \sum_{j=1}^{n_e} T_{ij} v_j$   
 192 and  $T_v \rightarrow T_{ij}$ , where  $T_{ij}$  is an  $n_e \times n_e$  matrix and  $v_j$  is a vector of length  $n_e$ . All numerical results  
 193 in this paper were obtained using a piecewise constant discretization of the stress change functions  
 194 by assuming that slip is constant over regularly spaced elements along the fault.



195 The stability condition given by equation (8) is only valid for  $2 \times 2$  matrices (e.g. *Luís*, 2021).  
 196 The eigenvalues must be explicitly calculated for numerical analysis (e.g. *Viesca*, 2016a,b; *Ray and*  
 197 *Viesca*, 2017; *Viesca*, 2023). The eigenvalues and eigenvectors can be directly computed using  
 198 standard numerical routines; here the MATLAB functions *eig* and *eigs* are used. Numerically  
 199 computing the eigenvalues will indicate if a fault system is stable or unstable for the specific set  
 200 of RSF, elastic, and geometrical parameters that define the system. Determining the conditions (if  
 201 any) where stability changes requires an iterative assessment of the stability for different parameter  
 202 values. In this paper, critical fault lengths are determined using a bisection method to locate values  
 203 of  $L$  where the stability changes (to within  $\pm d\xi/2$ ) while other properties are held constant.

### 204 3.2 Nonlinear Simulations

205 In Section 4.2 the results of a limited set of simulations of the full nonlinear governing equations  
 206 are presented to confirm some of the linear stability results. In these simulations the fault is loaded  
 207 such that the steady state slip velocity along the entire fault is equal to  $v_0 = 10^{-9}$  m/s. These  
 208 simulations use the aging law for state variable evolution and rather than equation (1), use the  
 209 regularized form of the rate and state friction equation (*Rice and Ben-Zion*, 1996; *Lapusta et al.*,  
 210 2000)

$$\mu(v, \theta) = a \sinh^{-1} \left[ \frac{v}{2v_0} \exp \left( \frac{\mu_0 + b \ln(v_0 \theta / d_c)}{a} \right) \right]. \quad (11)$$

211 In discrete form the stressing rate balance at the center of each fault element is

$$\dot{\mu}_i \sigma_i + \dot{\tau}_I = \sum_{j=1}^{n_e} T_{ij}(v_j - v_0) - \mu_i \sum_{j=1}^{n_e} N_{ij}(v_j - v_0), \quad (12)$$

212 where  $\dot{\mu}$  is found from equation (11) and  $\dot{\tau}_I$  is the radiation damping approximation for the inertial  
 213 stressing rate (*Rice*, 1993). The governing equations (12) with equation (11) and the aging law  
 214 were solved along the entire length of the fault using a boundary element method implemented in  
 215 MATLAB (see Data Availability statement for code availability).

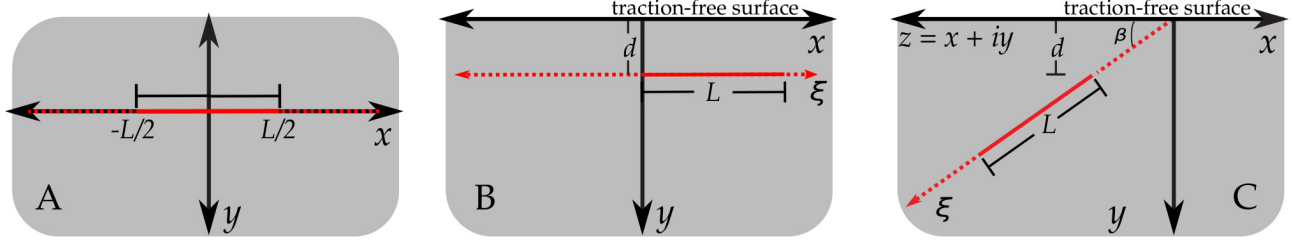


Figure 1: Diagrams of the fault system geometries used in this paper. In each panel a fault of length  $L$  is located by the solid red line; the dashed red line is the extension along the  $\xi$ -axis. **(A)** A fault in an infinite full-space, where the  $x$ - and  $\xi$ -axes coincide. **(B)** A fault at a depth  $d$ , parallel to a traction-free surface; both infinite and finite length systems are considered. **(C)** A fault dipping at an angle  $\beta$  relative to a traction-free surface, with its up-dip edge at a depth  $d$ . In panels B and C, the traction-free upper surface is defined by  $y = 0$ .

## 216 4 Results

217 Results are presented first for a finite length fault that is parallel to a traction-free surface, using  
 218 a thin-layer approximation for the stress change functions. Analytical results are obtained for the  
 219 thin-layer system, which provide insight into more complicated systems; numerical results for this  
 220 system are presented as well. Next results are obtained numerically for vertical strike-slip faults, as  
 221 well as faults in an infinite full-space. Finally, dip-slip faults of any orientation in a system with a  
 222 traction-free surface are examined.

### 223 4.1 Thin Layer Approximation

224 Consider a fault of length  $L$  that is parallel to a traction-free surface at a depth  $d$  (Figure 1B).  
 225 In general, this system will have a nonzero  $N(\xi, v)$  for in-plane sliding (see Section 4.3). However,  
 226 when  $d \ll L_b = d_c G' / (b\sigma_0)$  then  $N(\xi, v) = 0$  and the change in shear stress is (Viesca, 2016a)

$$T(\xi, v) = (dE') \frac{\partial^2 v}{\partial \xi^2}, \quad E' = \begin{cases} \frac{2G}{1-\nu}, & \text{in-plane sliding} . \\ G, & \text{anti-plane sliding} . \end{cases} \quad (13)$$

227 Note that equation (13) is a special case of the stress change function for a dipping fault geometry  
 228 illustrated in Figure 1C, as described in Section 4.3. The critical wavelength for an infinitely long

229 fault in this system is (Viesca, 2016b)

$$h_L^* = \frac{2\pi L_{bh}}{(1 - a/b)^{1/2}}, \quad (14)$$

230 where  $L_{bh} = \sqrt{dE'd_c/(b\sigma_0)}$  (see Appendix B.3.1).

231 Due to the simplicity of equation (13), analytical results for the critical fault length and the  
 232 wavelengths of unstable modes can be obtained for finite length faults in this system. By assuming  
 233 a solution for  $v(\xi, t)$  of the form of equation (10), the normalized shear stress change function  
 234 becomes  $\hat{T}_{\hat{v}}/b = -(n\pi L_{bh}/L)^2$  (see Appendix B.3.2 for details). Then via equation (8) the instability  
 235 condition for the fault length becomes

$$L > \frac{n\pi L_{bh}}{(1 - a/b)^{1/2}} = \frac{nh_L^*}{2}. \quad (15)$$

236 Since the right hand side of equation (15) is smallest at  $n = 1$ , the critical fault length is

$$L_L^* = \frac{h_L^*}{2}. \quad (16)$$

237 Equation (16) indicates that the fault becomes unstable when it is long enough that the wavelength  
 238  $\lambda$  of the first mode ( $n = 1$ ) of equation (10) becomes equal to  $\lambda = 2\pi/k = 2L = h_L^*$ .

239 The critical fault length  $L_L^*$  can also be numerically determined using the method described in  
 240 Section 3.1.2. In this case by choosing a value of  $a/b$  then computing the stability of the system for  
 241 different values of  $L$ . Then the critical fault length coincides with the value of  $L$  where the stability  
 242 changes. Figure 2A displays the results of this process for nine different values of  $a/b$  and shows  
 243 that the numerically determined values of  $L_L^*$  agree with equation (16).

244 As the fault length increases above  $L_L^*$ , progressively higher mode numbers will become unstable  
 245 and the wavelength of the highest unstable mode number will approach  $h_L^*$  as  $L \rightarrow \infty$ . From  
 246 equation (15), the total number of unstable modes that a fault can host is  $n_T = \text{Fl}(2L/h_L^*)$ , where  
 247  $\text{Fl}(q)$  gives the greatest integer less than or equal to some quantity  $q$ . The wavelength of the highest  
 248 mode number  $n_T$  is

$$\lambda_{n_T} = \frac{2L}{\text{Fl}(2L/h_L^*)}, \quad \text{and} \quad \lim_{L \rightarrow \infty} \lambda_{n_T} = h_L^*. \quad (17)$$

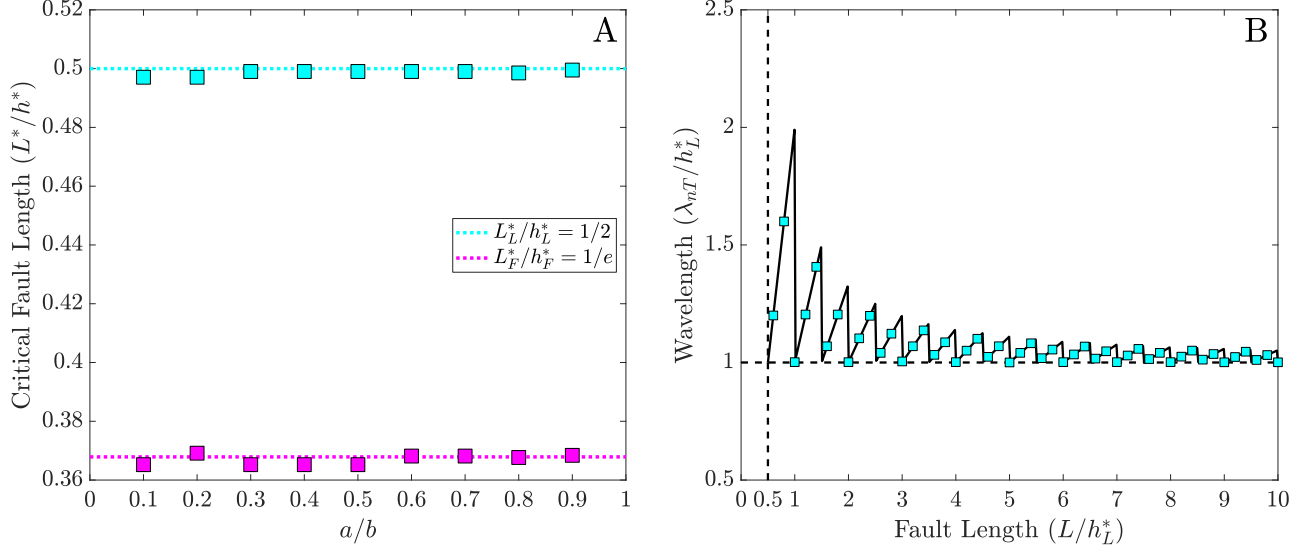


Figure 2: **(A)** Critical faults lengths  $L^*$  normalized by  $h_L^*$  for the thin layer (cyan) and by  $h_F^*$  for strike-slip/full-space (magenta) systems. The dotted lines correspond to expressions as shown in the legend; the squares are numerically determined boundaries as described in Section 3.1.2. Grid spacing for numerical calculations:  $d\xi = L_b/80$  for full-space and  $d\xi = L_{bh}/80$  for thin layer. **(B)** Wavelengths of the highest unstable mode number as a function of the fault length for the thin layer system, for  $a/b = 0.5$ . Both the wavelengths and the fault lengths are normalized by the critical wavelength  $h_L^*$ . The black line shows the analytical result given by equation (17), the cyan squares show the numerically determined wavelengths. For all calculations  $L_b = 1600$  km; for thin layer calculations  $d = 0.01L_b$ .

249 Equation (17) predicts that  $\lambda_{nT} \geq h_L^*$  and approaches  $h_L^*$  with a type of saw-tooth pattern as  
 250  $L \rightarrow \infty$  (Figure 2B). This result can also be confirmed numerically by computing the wavelength  
 251 of the eigenvector for the highest unstable mode as a function of the fault length for  $L > L_L^*$   
 252 (Figure 2B). The close agreement between the analytical and numerical analyses both validates the  
 253 numerical method and confirms the behavior for finite length faults.

254 Another important consequence of finite fault length is that there is no minimum failure patch  
 255 length required to generate an unstable sliding event for faults longer than  $L_L^*$ . Since equation (10)  
 256 is a superposition of all mode numbers, instability will occur if any mode has a positive growth  
 257 rate  $p$ . Thus, any set of initial conditions that gives  $A_n \neq 0$  for an unstable mode will generate an  
 258 instability; and there are no conditions on the length scale of the perturbation. This is illustrated  
 259 further in the next section.

## 260 4.2 Vertical Strike-Slip Faults in a Half-Space and Full-Space Faults

261 Now consider a fault of length  $L$  embedded in a homogeneous full-space (Figure 1A). For this  
 262 system  $N(\xi, v) = 0$  and the change in shear stress is given by (e.g. *Segall*, 2010)

$$T(\xi, v) = \frac{G'}{2\pi} \int_{-L/2}^{L/2} \frac{\partial v / \partial s}{s - \xi} ds. \quad (18)$$

263 This stress change function is also valid for a vertical strike-slip fault in a half-space, in which  
 264 case the integration is taken over  $[d, d + L]$  and  $G' = G$  (Figure 1C with  $\beta = 90^\circ$ ). Equation (18)  
 265 takes the form of a Hilbert transform for an infinitely long fault ( $L \rightarrow \infty$ ). Then the nucleation  
 266 wavelength  $h_F^*$  given by equation (3) can be obtained from equation (8) after applying a Fourier  
 267 transform (see Appendix B.2).

268 Analytical analysis using Fourier transforms cannot be applied to finite length faults due to  
 269 the finite integration interval in equation (18). Instead, the stability analysis can be conducted  
 270 numerically in the same manner as for the thin layer system, using the method described in Section  
 271 3.1.2. The results of the numerical stability analysis (Figure 2A) show that the critical fault length  
 272 for the full-space system is

$$L_F^* \approx h_F^* / e. \quad (19)$$

273 Equation (19) is an approximate equality in the absence of analytical results. The scaling with  $e^{-1}$   
 274 is an interesting feature of equation (19) that emerges from the numerical linear stability analysis.  
 275 A mathematical explanation for this scaling would require obtaining analytical results that in turn  
 276 would require finding an exact or approximate solution to equation (18) after assuming a solution  
 277 for  $v(\xi, t)$  of the form of equation (10).

278 However, equation (19) is also supported by simulations of the full nonlinear governing equations  
 279 following Section 3.2. Figure 3 shows the results of six sets of simulations using three values of  $a/b$   
 280 and two values of  $L_b$ . Nine simulations, each with a different fault length, were run for each pair  
 281 of  $(a/b, L_b)$  values. In these simulations the initial conditions were set to the uniform steady state  
 282 values, except for one element at the center of the fault where  $v(\xi = 0, t = 0) = 0.99v_0$ . Hence the  
 283 spatial extent of the initial perturbation is as small as the numerical discretization allows. Three  
 284 additional sets of simulations for  $L_b = 1600$  km were conducted with the perturbation applied to a

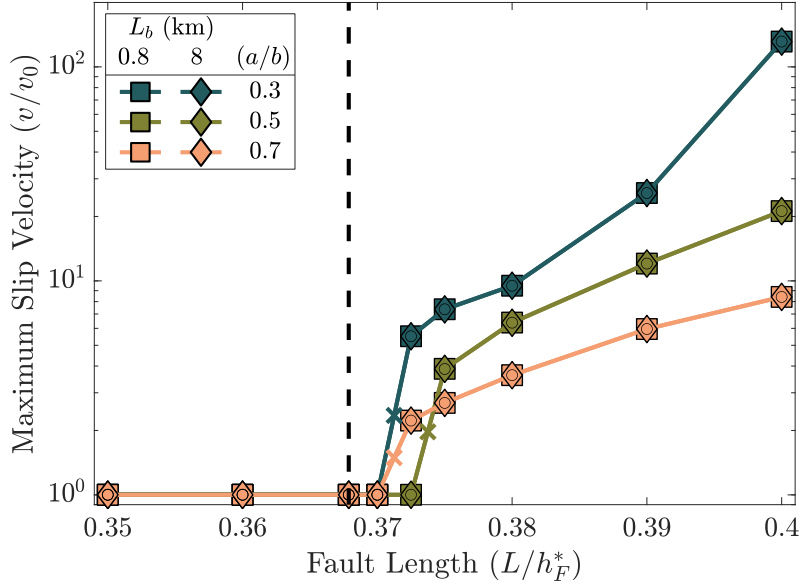


Figure 3: Normalized maximum slip velocities as a function of normalized fault length for faults in a full-space, or vertical strike-slip faults in a half-space. Colors correspond to values of  $a/b$  and symbols to values of  $L_b$  as indicated in the legend. Each symbol corresponds to an individual simulation. The squares and diamonds show results from simulations where a single numerical element at the center of the fault was perturbed; the circles show results where an element at the edge of the fault was perturbed. Note that the symbols overlie each other for each value of  $a/b$ , so there is no dependence on  $L_b$  or the location of the perturbation. The approximate critical fault lengths are marked by crosses and the vertical black, dashed line indicates  $L/h_F^* = e^{-1}$ . Grid spacing for simulations:  $d\xi = L_b/80$ .

285 single element at the edge of the fault.

286 The simulations were run until either consistent oscillations of maximum slip rate on the fault  
 287 developed (i.e. a limit cycle), or the sliding velocity reached a uniform steady state such that  
 288  $v(\xi, t) = v_0$ . The critical fault length for each pair of  $(a/b, L_b)$  values lies in the interval of fault  
 289 lengths that separate growth and decay of the initial perturbation, as indicated by the maximum  
 290 slip velocity. These critical fault lengths (normalized by  $h_F^*$ ) are  $0.37125 \pm 0.00125$  for  $a/b = 0.3, 0.7$   
 291 and  $0.37375 \pm 0.00125$  for  $a/b = 0.5$ . There is no dependence on the value of  $L_b$  or the location  
 292 of the perturbation (Figure 3). These critical fault lengths are within 2% of the value given by  
 293 equation (19). Since the perturbation was restricted to a single fault element, these results also  
 294 indicate that there is no minimum perturbation length scale.

### 295 4.3 Dip-Slip Faults

296 Consider in-plane sliding on a fault that is dipping at an angle  $\beta$  relative to the traction-free  
 297 surface of a homogeneous, elastic half-space (Figure 1C). The up-dip edge of the fault is buried at a  
 298 depth  $d$  below the traction-free surface. Both the full-space and parallel fault geometries are special  
 299 cases of this dipping fault geometry. The full-space geometry is obtained when  $d \rightarrow \infty$ , and the  
 300 parallel fault geometry is obtained when  $d \neq 0$  and  $\beta = 0$ .

301 Stress change functions for the half-space geometry are available in the literature (*Dmowska and*  
 302 *Kostrov, 1973; Freund and Barnett, 1976; Rudnicki and Wu, 1995*), and can be written as

$$T(\xi, v) = \int_l^{l+L} \Psi(z, \beta) \frac{\partial v}{\partial s} ds, \quad (20)$$

303 where  $l = d/\sin(\beta)$ , and  $\Psi(z, \beta)$  is an analytic function of the complex variable  $z = x + iy$  (*England,*  
 304 *2003*). A similar expression holds for  $N(\xi, v)$ . A derivation of these functions is presented in  
 305 Appendix C. Note that these stress change functions are equivalent to using the *Okada (1992)*  
 306 solutions for the middle of a very long dip-slip fault (e.g. *Liu and Rice, 2007*).

#### 307 4.3.1 Velocity-Weakening Behavior

308 Critical fault lengths  $L_D^*$  for the dipping geometry can be determined by choosing a burial depth  
 309  $d$  and dip angle  $\beta$  and then conducting a numerical stability analysis as described in Section 3.1.2.  
 310 Changing the value of  $L$  for fixed values of  $d$  and  $\beta$  corresponds to changing the down-dip depth of  
 311 the fault. The stability calculation was repeated for dip angles in the range  $\beta = 0^\circ - 90^\circ$  and burial  
 312 depth values  $d/h_F^* = 0, 10^{-3}, 10^{-2}, 10^{-1}, 1$  (the value  $d = 0$  was omitted for  $\beta = 0^\circ$ ). This process  
 313 was carried out for values of  $\mu_0 = 0.2, 0.6, 1$ , for both thrust and normal faults (Figure 4).

314 The critical fault length  $L_D^*$  approaches the full-space value given by equation (19) as  $d \rightarrow h_F^*$ .  
 315 Therefore  $L_D^* = L_F^*$  at depths  $d \geq h_F^*$  and Figure 4 shows critical fault lengths for both thrust  
 316 and normal faults in any possible orientation. For burial depths  $d < h_F^*$ , the critical fault length is  
 317 approximately a log-linear function of  $d$  (Figure 5A).

318 The critical fault length  $L_D^*$  on the dip angle in a manner that is different for thrust and normal  
 319 faults. There is also a secondary dependence on the value of  $\mu_0$  that depends on the sense of slip.  
 320 For both thrust and normal faults,  $L_D^*$  increases with dip angle up to a value of  $20^\circ - 40^\circ$ , depending

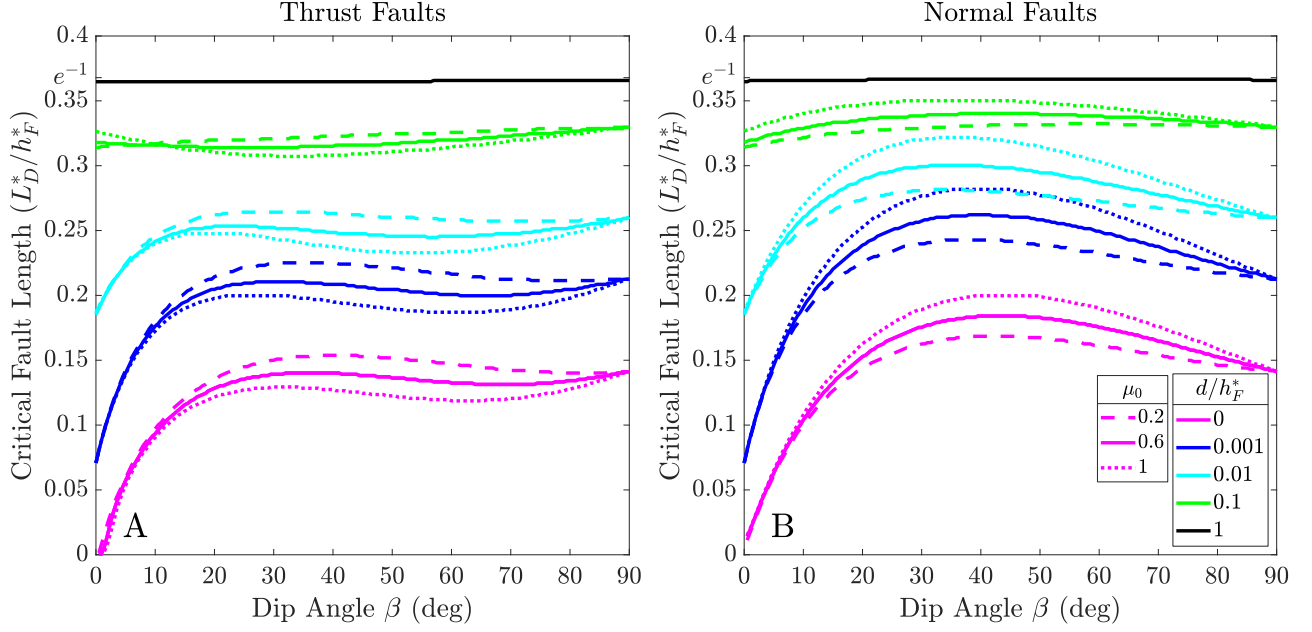


Figure 4: Critical fault lengths  $L_D^*$  for thrust and normal faults as a function of dip angle  $\beta$ , burial depth  $d$ , and friction coefficient  $\mu_0$ . Critical fault lengths and burial depths are normalized by  $h_F^*$ . Values of  $d$  are indicated by colors, and values of  $\mu_0$  by line styles as indicated in the legend. Since the critical fault length can be very small, for these calculations the grid spacing was set to  $d\xi = L_b/80$  or  $d\xi = L/250$ , whichever is smaller. The solid black lines are equal to  $e^{-1}$  to within 0.8%.

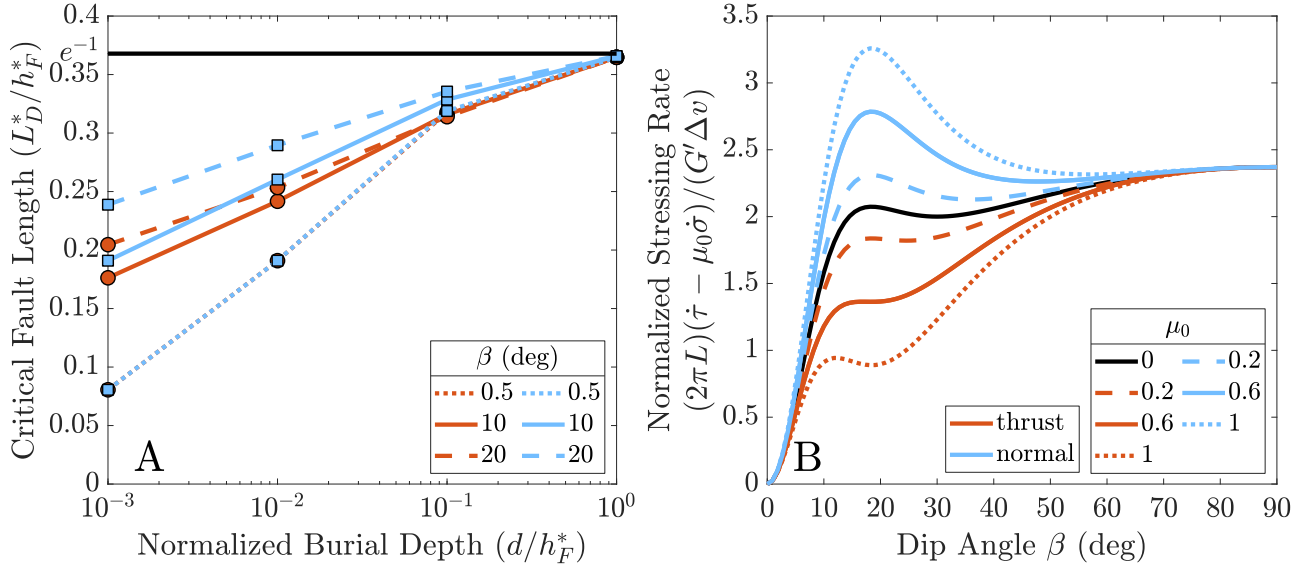


Figure 5: **(A)** Examples of critical fault lengths for thrust and normal faults as a function of burial depth for  $\beta = 0.5^\circ, 10^\circ, 20^\circ$ . **(B)** Normalized stressing rates  $(2\pi L)(\dot{\tau} - \mu_0 \dot{\sigma}_E)/(G' \Delta v)$  for thrust and normal faults. For both panels the sense of slip is indicated by colors as shown in the legend in panel B.



321 on the burial depth and sense of slip. For thrust faults,  $L_D^*$  then decreases to a secondary minimum  
 322 before increasing again as  $\beta \rightarrow 90^\circ$ . For normal faults,  $L_D^*$  reaches a maximum value then decreases  
 323 as  $\beta \rightarrow 90^\circ$ . Increasing the value of  $\mu_0$  decreases  $L_D^*$  for thrust faults, and does the opposite for  
 324 normal faults. Values of  $L_D^*$  can become quite small on shallowly dipping faults that are near  
 325 to the traction-free surface. In particular, as  $\beta \rightarrow 0^\circ$  on faults that break the surface ( $d = 0$ ),  
 326  $L_D^*/h_F^* \rightarrow 10^{-2}$  on normal faults and appears to approach zero on thrust faults.

327 The dependence of  $L_D^*$  on  $\beta$  and  $\mu_0$  can mostly be explained by considering the on-fault stressing  
 328 rates due to a uniform slip velocity distribution  $\Delta v$  on a dipping fault of length  $L$  with burial depth  
 329  $d = 0$ . The elastic stressing rate on the fault is  $\dot{\tau}_E - \mu\dot{\sigma}_E$  (see Section 3.1, Step 1), which can be  
 330 computed by evaluating the stress change functions at the center of the fault  $\xi = L/2$  (e.g. *Kato*  
 331 *and Hirasawa, 1997*). The stressing rate has a dependence on  $\beta$  and  $\mu_0$  that shares some of the  
 332 same features as that of  $L_D^*$ ; including similar behavior as  $\beta \rightarrow 0^\circ$  and  $\beta \rightarrow 90^\circ$ , and the same style  
 333 of dependence on  $\mu_0$  for thrust and normal faults (Figure 5B).

334 The stressing rate calculation also provides an explanation for why values of  $L_D^*$  become very  
 335 small at shallow dip angles. Sliding instability develops when the frictional weakening rate  $\dot{\mu}$  is  
 336 greater than the elastic stressing rate. The elastic stressing rate is approximately proportional  
 337 to  $\beta/L$  for dip angles less than about  $10^\circ - 20^\circ$  (Figure 5B). Then for a given set of frictional  
 338 parameters, when the dip angle is small only shorter length faults can relieve elastic stress faster  
 339 than the frictional weakening rate. This leads to the results displayed in Figure 4.

### 340 4.3.2 Velocity-Strengthening Behavior

341 As noted in the Introduction, it is possible for unstable behavior to occur on velocity-strengthening  
 342 faults when a coupling between slip and normal stress exists, i.e. when  $N(\xi, v) \neq 0$ . The parameter  
 343 space for the dipping fault geometry is large; the stability behavior can be expected to depend  
 344 on frictional and elastic parameters  $\mu_0$ ,  $a/b$ ,  $L_b$ ; burial depth  $d$ ; dip angle  $\beta$ ; and fault length  $L$ .  
 345 Additionally, while normalization by  $h_F^*$  accounts for dependence on RSF and elastic parameters  
 346 for velocity-weakening behavior,  $h_F^*$  does not exist on velocity-strengthening faults. Therefore the  
 347 results in this section are restricted to an infinitely long fault that is parallel to a traction-free  
 348 surface, which reduces the parameter space to  $\mu_0$ ,  $a/b$ , and a normalized burial depth  $d/L_b$ . In this  
 349 case equation (7) can be used to determine the stability of the system (see Appendix B.4 for stress

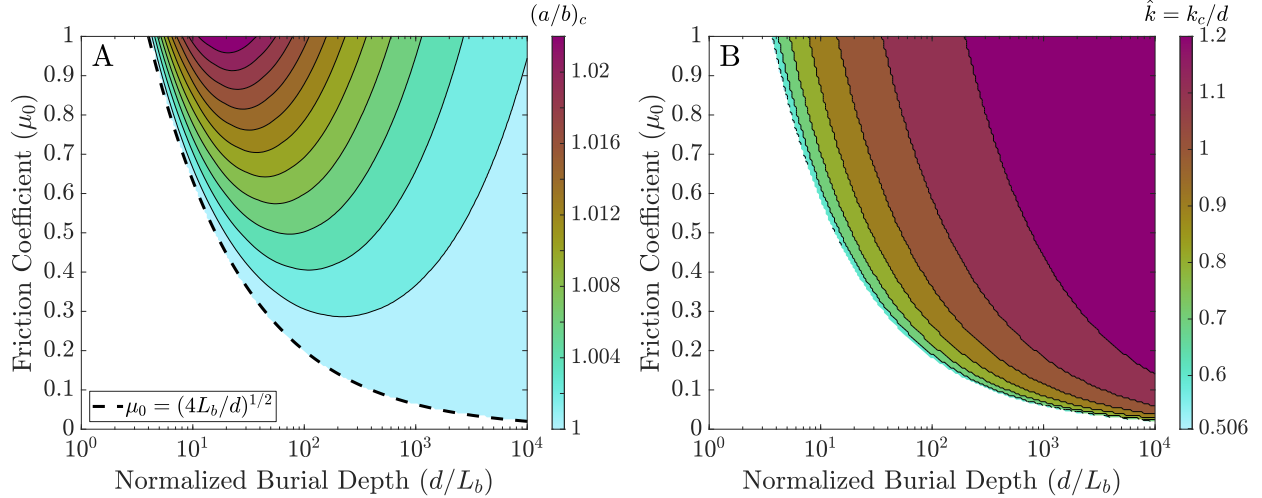


Figure 6: **(A)** Values of  $(a/b)_c$ , and **(B)**  $k_c/d$  for unstable behavior on infinitely long, velocity-strengthening faults near a traction-free surface, as a function of friction coefficient  $\mu_0$  and normalized burial depth  $d/L_b$ . The black, dashed line in **(A)** corresponds to the shallow stability boundary given by  $d = (2/\mu_0)^2 L_b$ .

350 change functions and details)

351 Figure 6 shows the results of choosing values of  $\mu_0$  and  $d/L_b$ , then determining the maximum  
 352 value of  $(a/b)_c = (a/b) > 1$  that satisfies  $\text{Re}(p) > 0$  in equation (7). One striking feature of the  
 353 results is that unstable behavior only exists at depths greater than some minimum value that is  
 354 very well approximated by

$$d = (2/\mu_0)^2 L_b . \quad (21)$$

355 The details of obtaining equation (21) are provided in Appendix B.4. At shallower depths there are  
 356 no unstable solutions to equation (7) for  $(a/b) > 1$ . This shallow, stable region is not related to the  
 357 thin layer limit that occurs at  $d/L_b \ll 1$ . Where unstable behavior occurs, for constant  $\mu_0$  there is  
 358 depth at which  $(a/b)_c$  reaches a maximum value. While for constant  $d/L_b$ , values of  $(a/b)_c$  increase  
 359 monotonically with  $\mu_0$ , so the velocity-strengthening instability is enhanced when friction is higher.

360 An extensive parameter study to determine the effects of finite fault length and dip angle is  
 361 beyond the scope of this study. However, some insight can be gained by examining the critical  
 362 wavelengths that correspond to the values of  $(a/b)_c$ . Each value of  $(a/b)_c$  shown in Figure 6A  
 363 occurs at some critical wavenumber  $k_c$  that is in the neighborhood of  $k_c/d \approx 1$  regardless of the  
 364 value of  $(a/b)_c$  (Figure 6B). By analogy with the velocity weakening results, if  $L^* \approx h^*/e = 2\pi/(ek_c)$ ,  
 365 then for any set of values  $[\mu_0, (a/b)_c, d/L_b]$  taken from Figure 6A, the fault length would have to be

366  $L \geq 2\pi d/e$  for unstable behavior to occur.

## 367 5 Discussion

### 368 5.1 Some Theoretical Considerations

369 A main result in this paper is that the linear stability of frictional sliding depends on overall  
370 fault length. The critical fault length  $L^*$  can replace the concept of the critical nucleation length  
371 represented by  $h^*$ . Velocity-weakening faults are unstable if they are longer than  $L^*$ . In terms of  
372 linear behavior (where deviations from steady sliding are small), there is no minimum perturbation  
373 length scale that is needed to trigger an unstable sliding event if the fault length is longer than  $L^*$ .  
374 Velocity-weakening faults that are shorter than  $L^*$  should be considered conditionally stable, in that  
375 large perturbations out of the linear regime could generate unstable sliding events (e.g. *Gu et al.*,  
376 1984). For vertical strike-slip faults  $L^* \approx h_F^*/e$  (Section 4.2), while for dip-slip faults the critical  
377 fault length is a function of the dip angle and burial depth (Section 4.3). For sliding systems that  
378 can be treated as a thin layer (e.g. landslides, glaciers, or ice streams)  $L^* = h_L^*/2$ .

379 After nucleating, the details of how an instability develops (and any related length scales) depend  
380 on the nonlinear governing equations (*Rubin and Ampuero*, 2005, 2009; *Ampuero and Rubin*, 2008;  
381 *Viesca*, 2016a,b; *Ray and Viesca*, 2017; *Viesca*, 2023). As velocities increase towards inertially  
382 limited values on faults that obey the aging law, sliding localizes to patches with lengths that scale  
383 with  $L_b$  (e.g. *Rubin and Ampuero*, 2005; *Viesca*, 2016a). While for the slip law, *Viesca* (2023)  
384 showed that accelerating slip localizes towards a point, so that there is no minimum patch length.  
385 Together with the results in this paper, the implication is that there is no nucleation length scale on  
386 faults that obey the slip law. This could be important, considering that recent work has shown that  
387 the slip law can explain a wider range of experimental observations than the aging law (*Bhattacharya*  
388 *et al.*, 2015, 2017, 2022).

389 However, all of the results in this paper rely on idealized fault systems that do not include  
390 multiple interacting faults; heterogeneous frictional and material properties; non-uniform steady  
391 state conditions; or inelastic deformation. It is possible that examining more realistic finite length  
392 fault systems may lead to different conclusions regarding nucleation length scales. In addition,  
393 the results also represent the idealizations that are incorporated into the RSF equations as they

394 are applied to laboratory experiments. The RSF equations, including the multiple different state  
 395 evolution laws, have all been determined through application of spring-slider models to experimental  
 396 data (*Dieterich, 1979; Ruina, 1983*). Thus the possible effects of traction-free surfaces (which are  
 397 numerous in most experimental geometries) on laboratory frictional behavior is mostly unknown  
 398 (*Aldam et al., 2016*).

399 Finally, it is clear that when examining heterogeneous systems, the fault system must be treated  
 400 as a single entity. For example, applying the critical nucleation length from equation (3) to the  
 401 velocity-weakening sections of a dipping fault that also has velocity-strengthening sections will  
 402 result in inaccurate assessments of sliding stability. The stability behavior will instead depend on  
 403 the geometrical aspects as well as the frictional properties in both the velocity-weakening and -  
 404 strengthening portions of the fault (*Skarbek et al., 2012; Dublanche et al., 2013; Ray and Viesca,*  
 405 *2017; Yabe and Ide, 2017; Luo and Ampuero, 2018*).

## 406 5.2 Some Practical Considerations

407 Proximity to a traction-free surface, as measured by  $h_F^*$  or  $L_b$ , has a significant influence on  
 408 stability properties. Since both  $h_F^*$  and  $L_b$  are inversely proportional to effective normal stress, the  
 409 normalized burial depths in Figures 4 and 6 are smaller on faults with high pore fluid pressure. This  
 410 means that the influence of the free surface is enhanced on overpressured fault systems. High pore  
 411 pressure leads to smaller normalized critical fault lengths, but larger values of  $h_F^*$ . If the burial depth  
 412 is less than  $h_F^*$ , then the free surface will influence the stability behavior. This effect should for  
 413 example be important in the shallow regions of subduction zones and in areas of induced seismicity  
 414 where pore pressures can be elevated. Particularly on subduction megathrust plate boundaries, the  
 415 combination of shallow dip angles and high pore pressures should lead to very small normalized  
 416 critical fault lengths.

417 The effect of shallow burial depth on unstable behavior for velocity-strengthening faults is more  
 418 complicated. A parallel fault should be the most unstable geometry for a nonzero burial depth  $d$ ,  
 419 since on a dipping fault the depth from the traction-free surface will increase with down-dip distance.  
 420 The values of  $(a/b)_c$  for the infinite fault system in Figure 6A are close to velocity-neutral, so it  
 421 seems reasonable to assume that values of  $(a/b)_c$  would be even closer to unity on finite length,  
 422 dipping faults that are buried. However, the velocity-weakening results show that intersecting

423 the free surface causes a significant reduction in stability;  $L_D^*/h_F^*$  decreases logarithmically with  
424 decreasing  $d/h_F^*$ . So it is possible that values of  $(a/b)_c$  may be larger on dipping faults where  $d = 0$ .  
425 Certainly more work is needed to understand this behavior.

426 Multiple effects have been described that can cause unstable sliding on velocity-strengthening  
427 faults: contrasting elastic parameters across a fault (*Rice et al.*, 2001; *Ranjith*, 2014); poroelasticity  
428 (*Heimisson et al.*, 2019); “fault valve” behavior (*Ozawa et al.*, 2024); and proximity to a traction-free  
429 surface (this paper; *Aldam et al.*, 2016). All of these features are commonplace in fault systems as  
430 well as in other frictional systems like landslides and ice streams. For example, all of these effects  
431 could be present in the shallow regions of subduction zones and may contribute towards enabling  
432 shallow slow slip events (e.g. *Saffer and Wallace*, 2015), or influencing the behavior of tsunami  
433 earthquakes (e.g. *Bilek and Lay*, 2002).

## 434 6 Conclusion

435 The results in this paper show how even simple types of geometrical complexity can change  
436 stability behavior. Using numerical methods makes it possible to conduct linear stability analyses  
437 for a wide range of fault systems that cannot be examined using analytical techniques. Some  
438 examples of systems for which stress change functions are available in the literature are multi-fault  
439 systems and non-planar faults in a 3D homogeneous elastic half-space (*Okada*, 1992; *Meade*, 2007).  
440 Functions are also available for different types of viscoelastic geometries (*Segall*, 2010; *Lambert and*  
441 *Barbot*, 2016, e.g.). Heterogeneous on-fault frictional properties can be used with any existing stress  
442 change functions (e.g. *Ray and Viesca*, 2017). Finally, numerical stability methods could also be  
443 extended to include dilatancy and changes in pore pressure, or other types of frictional constitutive  
444 behavior (e.g. *Segall and Rice*, 1995; *Chen and Spiers*, 2016; *Barbot*, 2022).

## 445 Data Availability

446 All of the calculations and figures in this paper can be reproduced using a MATLAB package *RS-*  
447 *FaultZ* available at <https://github.com/rmskarbek/RSFaultZ> (*Skarbek*, 2024). The m-files for au-  
448 tomatically generating figures are stored in the github repository directory: `RSFaultZ/examples/stability`.

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453 2019; *Crameri et al.*, 2020).

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## 597 Appendix A Linearization of RSF Equations

598 Additional mathematical details are provided here for obtaining the Jacobian matrix given by  
 599 equation (6). First, using equations (2) and (4), the linearized equations can be written as

$$\dot{v} = \left( \frac{\partial F_0}{\partial v} \right) v + \left( \frac{\partial F_0}{\partial \theta} \right) \theta = \left( \frac{v_0}{a} \right) \left[ \frac{1}{\sigma_0} \left( \frac{\partial \dot{\tau}_E}{\partial v} - \mu_0 \frac{\partial \dot{\sigma}_E}{\partial v} \right) + \frac{b}{d_c} \right] v + \left( \frac{bv_0^3}{ad_c^2} \right) \theta, \quad (\text{A1})$$

600 and

$$\dot{\theta} = \left( \frac{\partial H_0}{\partial v} \right) v + \left( \frac{\partial H_0}{\partial \theta} \right) \theta = - \left( \frac{1}{v_0} \right) v - \left( \frac{v_0}{d_c} \right) \theta. \quad (\text{A2})$$

601 Equations (A1) and (A2) can be used to define a dimensional Jacobian. The elements of equation  
 602 (6) are obtained after changing to the dimensionless variables defined by  $\hat{t} = (v_0/d_c)t$ ,  $\hat{v} = v/v_0$ ,  
 603 and  $\hat{\theta} = (v_0/d_c)\theta$ . Dimensionless stress change functions are obtained by normalizing stresses by  
 604  $\sigma_0$ . So for example,  $\dot{\tau}_E = T(\xi, v) = (\sigma_0 v_0/d_c)\hat{T}$ .

## 605 Appendix B Analytical Linear Stability Results

### 606 B.1 Spring-Slider

607 The shear stress change in the basic spring-slider model is

$$\dot{\tau}_E = K(v_0 - v), \quad (\text{B1})$$

608 where  $K$  is a normalized spring stiffness with units of [Stress / Length]. Using the same dimensionless  
 609 variables defined in A, the dimensionless shear stress change function is

$$\hat{T}(\hat{v}) = \frac{d_c K}{\sigma_0} (1 - \hat{v}). \quad (\text{B2})$$

610 Inserting the derivative of equation (B2) with respect to  $\hat{v}$  into equation (8) and setting the left-hand  
 611 side equal to zero yields the critical stiffness  $K_c = \sigma_0(b - a)/d_c$ .

## 612 B.2 Infinite Fault in a Full-Space

613 For infinite faults the critical wavelength can be found by searching for solutions of the form  
 614  $v(\xi, t) = A \exp(pt + ik\xi)$ . For a full-space, the shear stress change function can be found by  
 615 substituting this expression into equation (18), for  $L \rightarrow \infty$ ; this is essentially the method used by  
 616 (*Rice et al.*, 2001):

$$T(\xi, v) = ikA \left( \frac{G'}{2\pi} \right) \int_{-\infty}^{\infty} \frac{\exp(pt + iks)}{s - \xi} ds . \quad (\text{B3})$$

617 After making a change of variables  $u = s - \xi$ , equation (B3) becomes

$$\begin{aligned} T(\xi, v) &= ik \left( \frac{G'}{2\pi} \right) A \exp(pt + ik\xi) \int_{-\infty}^{\infty} \frac{\exp(iku)}{u} du \\ &= - \left( \frac{|k|G'}{2} \right) v , \end{aligned} \quad (\text{B4})$$

618 where the integral in the first line is a Fourier Transform of  $1/u$  and is equal to  $i\pi \text{sgn}(k)$ . Us-  
 619 ing the previously defined dimensionless variables, but leaving  $k$  in dimensional form, the critical  
 620 wavenumber  $k_c$  from equation (8) is

$$\left( \frac{b}{a} \right) \left( 1 - \frac{L_b |k_c|}{2} \right) - 1 = 0 , \quad (\text{B5})$$

621 which leads to equation (3) since the critical wavelength is defined as  $h_F^* = \lambda_c = 2\pi/k_c$ .

## 622 B.3 Thin Layer

### 623 B.3.1 Infinite fault

624 The critical wavelength for the thin layer system can be found by following the same procedure  
 625 for the full-space system, but using equation (13) for the shear stress change function.

$$T(\xi, v) = (dE') \frac{\partial^2}{\partial \xi^2} [A \exp(pt + ik\xi)] = -dE' k^2 v . \quad (\text{B6})$$

626 Using the dimensionless variables as before, equation (8) becomes

$$\left( \frac{b}{a} \right) [1 - (L_{bh} k_c)^2] - 1 = 0 , \quad (\text{B7})$$

627 with  $L_{bh} = \sqrt{dE'd_c/(b\sigma_0)}$  (Viesca, 2016b). Solving equation (B7) for the critical wavelength leads  
 628 to equation (14) for  $h_L^*$ .

### 629 B.3.2 Finite Fault

630 For a finite fault the deviation of the sliding velocity from steady state takes the form  $v(\xi, t) - v_0 =$   
 631  $a_n e^{pt} \sin(n\pi\xi/L)$ . After substituting this into equation (13), the shear stress change becomes

$$T(\xi, v) = -dE' \left(\frac{n\pi}{L}\right)^2 a_n e^{pt} \sin(n\pi\xi/L) = -dE' \left(\frac{n\pi}{L}\right)^2 v, \quad (\text{B8})$$

632 such that

$$T_v = \frac{\partial}{\partial v} T(\xi, v) = -dE' \left(\frac{n\pi}{L}\right)^2. \quad (\text{B9})$$

633 Equation (B9) can be normalized using the dimensionless quantities defined in Appendix A and  
 634 remembering that  $T$  has units of [stress/time], then

$$\frac{\hat{T}_v}{b} = - \left(\frac{n\pi L_{bh}}{L}\right)^2, \quad (\text{B10})$$

635 as in Section 4.1. Finally, the critical fault length is obtained by substituting equation (B10) into  
 636 equation (8), which yields equation (15).

### 637 B.4 Velocity-Strengthening Layer

638 The stress change functions for in-plane sliding on an infinitely long fault that is parallel to a  
 639 traction-free surface at a depth  $d$  are (e.g. Viesca, 2016a)

$$T(\xi, v) = \frac{G}{2\pi(1-\nu)} \int_{-\infty}^{\infty} \left\{ \frac{1}{s-\xi} - \frac{s-\xi}{4d^2 + (s-\xi)^2} + \frac{8d^2(s-\xi)}{[4d^2 + (s-\xi)^2]^2} \right. \\ \left. + \frac{4d^2(s-\xi)^3 - 48d^4(s-\xi)}{[4d^2 + (s-\xi)^2]^3} \right\} \frac{\partial v}{\partial s} ds, \quad (\text{B11})$$

640 and

$$N(\xi, v) = \frac{G}{2\pi(1-\nu)} \int_{-\infty}^{\infty} \left\{ \frac{32d^5 - 24d^3(s-\xi)^2}{[4d^2 + (s-\xi)^2]^3} \right\} \frac{\partial v}{\partial s} ds. \quad (\text{B12})$$

641 The stability of this system is most easily determined after applying a Fourier transform. Using the  
 642 Fourier transform pair:

$$\tilde{f}(k) = F[f(x)] = \int_{-\infty}^{\infty} f(x)e^{-ikx} dx \quad (\text{B13})$$

$$f(x) = F^{-1}[\tilde{f}(k)] = \frac{1}{2\pi} \int_{-\infty}^{\infty} \tilde{f}(k)e^{ikx} dx, \quad (\text{B14})$$

643 equations (B11) and (B12) become

$$\tilde{T}(k, \tilde{v}) = - \left( \frac{G'|k|}{2} \right) \left\{ 1 - e^{-2d|k|} [1 - 2d|k| + 2(dk)^2] \right\} \tilde{v}, \quad (\text{B15})$$

644 and

$$\tilde{N}(k, \tilde{v}) = -iG'k(dk)^2 e^{-2d|k|} \tilde{v}. \quad (\text{B16})$$

645 where tildes denote transformed quantities. Note that these functions are provided by *Viesca*  
 646 (2016a) using a different transform pair.

647 The eigenvalues  $p$  can then be computed from equation (7) after defining  $\Gamma = (1/b)(\tilde{T}_{\tilde{v}} - \mu_0\tilde{N}_{\tilde{v}})$   
 648 using equations (B15) and (B16). Using the dimensionless variables, and also defining  $\hat{k} = dk$  yields

$$\Gamma = - \left( \frac{L_b}{d} \right) \left\{ \frac{|\hat{k}|}{2} \left[ 1 - e^{-2|\hat{k}|} (1 - 2|\hat{k}| + 2\hat{k}^2) \right] - i\mu_0\hat{k}^3 e^{-2\hat{k}} \right\}. \quad (\text{B17})$$

649 The resulting equation for  $p$  is complex and depends on the values of  $(a/b)$ ,  $(L_b/d)$ ,  $\mu_0$ , and the  
 650 dimensionless wavenumber  $\hat{k}$ . The results in Figure 6 were obtained through an iterative process by  
 651 solving for  $p$  numerically as a function of  $\hat{k}$  for chosen values of  $L_b/d$  and  $\mu_0$ . For each pair of values  
 652  $(L_b/d, \mu_0)$ ,  $p(\hat{k})$  was first determined for a value of  $(a/b) < 1$ , which guarantees that  $\text{Re}[p(\hat{k})] > 0$   
 653 for some value of  $\hat{k}$ ; numerical tests showed that the maximum value of  $p(\hat{k})$  occurs in the vicinity  
 654 of  $\hat{k} \approx 1$ . This process was then repeated for incrementally larger values of  $(a/b)$  until  $\text{Re}[p(\hat{k})] < 0$   
 655 for all values of  $\hat{k}$ , which determines the values of  $(a/b)_c$  shown in Figure 6A.

656 The minimum depth for unstable behavior can be approximately determined by solving for  $p$   
 657 for a specific value of  $\hat{k}$ . From the results in Figure 6B, the stability boundary occurs at  $\hat{k} \approx 0.5$ ,

658 so that

$$\Gamma = - \left( \frac{L_b}{d} \right) \left[ \left( \frac{2 - e^{-1}}{8} \right) - i\mu_0 \left( \frac{e^{-1}}{8} \right) \right]. \quad (\text{B18})$$

659 Additionally, that stability boundary occurs at  $(a/b) = 1$ . With these values of  $\hat{k}$  and  $(a/b)$ , equation  
660 (7) becomes

$$p^2 - \left( \frac{L_b}{d} \right) \left[ \left( \frac{2 - e^{-1}}{8} \right) - i\mu_0 \left( \frac{e^{-1}}{8} \right) \right] p + \left( \frac{L_b}{d} \right) \left[ \left( \frac{2 - e^{-1}}{8} \right) - i\mu_0 \left( \frac{e^{-1}}{8} \right) \right] = 0. \quad (\text{B19})$$

661 Now  $p$  can be solved for using a procedure described in *Rice et al.* (2001). First, Figure 6 indicates  
662 that for a constant value of  $\mu_0$ , the real part of  $p$  changes sign as  $d/L_b$  increases from zero. The  
663 sign change occurs at  $p = i\rho$ ; substituting this into equation (B19) yields

$$\left[ -\rho^2 - \left( \frac{\mu_0 e^{-1} L_b}{8d} \right) \rho + \frac{(2 - e^{-1})L_b}{8d} \right] - i \left( \frac{L_b}{d} \right) \left[ \left( \frac{2 - e^{-1}}{8} \right) \rho + \frac{\mu_0 e^{-1}}{8} \right] = 0. \quad (\text{B20})$$

664 Equation (B20) is satisfied when both its real and imaginary parts are equal to zero. Setting the  
665 real part equal to zero provides an equation for  $\rho$  in terms of  $\mu_0$  and  $(L_b/d)$ :

$$\rho = - \left( \frac{\mu_0 e^{-1} L_b}{16d} \right) \pm \frac{1}{2} \sqrt{\left( \frac{\mu_0 e^{-1} L_b}{8d} \right)^2 - \frac{(2 - e^{-1})L_b}{2d}}. \quad (\text{B21})$$

666 Finally, inserting equation (B21) into the imaginary part of equation (B20) and setting it equal to  
667 zero provides an equation for  $d/L_b$  as a function of  $\mu_0$ . The best way to execute this final step is  
668 using a symbolic math program. The solution is

$$\frac{d}{L_b} = \frac{1}{8e} \left[ \frac{(2e - 1)(1 - 2e)^2}{\mu_0^2} - 1 \right] + 1/4 \approx \left( \frac{2}{\mu_0} \right)^2. \quad (\text{B22})$$

## 669 Appendix C Dip-slip Faults

670 Consider an edge dislocation in a 2D homogeneous elastic body. The dislocation induces dis-  
671 placement and stress fields throughout the elastic body that can be represented in terms of two  
672 complex potentials,  $\omega(z)$  and  $\Omega(z)$ , that are analytic functions of  $z$  (e.g. *England*, 2003; *Bower*,  
673 2009). The complex coordinate  $z$  is defined as  $z = x + iy = re^{i\phi}$  where  $(r, \phi)$  are radial coordinates  
674 with  $\phi$  measured from the  $x$ -axis in the direction of the  $y$ -axis.



675 For the dipping fault system shown in Figure 1C, the fault is located at  $\beta_0 = \pi - \beta$  along  
 676  $l \leq r \leq l + L$ , where  $l = d/\sin(\beta)$ , (also note that  $\xi = r$ ). The stress change functions can be  
 677 obtained by considering a distribution of dislocations along the fault, and computing the shear and  
 678 normal stresses that these dislocations induce on the fault itself. The first and most important step  
 679 is to determine the complex potentials for a single dislocation placed at  $z_0 = r_0 e^{i\beta_0}$ , with Burger's  
 680 vector  $be^{i\beta_0} = b \cos(\beta_0) + ib \sin(\beta_0)$  (e.g. *Freund and Barnett, 1976*).

681 In the  $x$ - $y$  plane the stress and displacement fields are given by:

$$\sigma_x + \sigma_y = 2 \left[ \Omega'(z) + \overline{\Omega'(z)} \right] , \quad (\text{C1})$$

$$\sigma_y - i\sigma_{xy} = \Omega'(z) + \overline{\Omega'(z)} + z\overline{\Omega''(z)} + \omega'(z) , \quad (\text{C2})$$

$$2G(u_x + iu_y) = (3 - 4\nu)\Omega(z) - z\overline{\Omega'(z)} - \overline{\omega(z)} , \quad (\text{C3})$$

682 where primes denote derivatives with respect to  $z$ , and bars denote complex conjugates (e.g. Section  
 683 2.5 in *England, 2003*). The displacements are denoted by  $u_x, u_y$ ; the normal stresses by  $\sigma_x$  and  $\sigma_y$ ,  
 684 and  $\sigma_{xy}$  is the shear stress. The normal and shear stresses on the fault can be obtained in the radial  
 685 coordinate system, in which case the stresses are

$$\sigma_r + \sigma_\phi = 2 \left[ \Omega'(z) + \overline{\Omega'(z)} \right] , \quad (\text{C4})$$

$$\sigma_\phi - i\sigma_{r\phi} = \Omega'(z) + \overline{\Omega'(z)} + e^{-2i\phi} [z\overline{\Omega''(z)} + \omega'(z)] , \quad (\text{C5})$$

$$2G(u_r + iu_\phi) = e^{-2i\phi} [(3 - 4\nu)\Omega(z) - z\overline{\Omega'(z)} - \overline{\omega(z)}] . \quad (\text{C6})$$

686 For a half-space with a traction-free surface at  $y = 0, z = x$ , the potentials can be written as

$$\Omega(z) = \Omega_0(z) + \Omega_1(z) , \quad \omega(z) = \omega_0(z) + \omega_1(z) , \quad (\text{C7})$$

687 where  $\Omega_0(z)$  and  $\omega_0(z)$  are the potentials for a full-space, and so will produce tractions along  $z = x$ ;  
 688 while  $\Omega_1(z)$  and  $\omega_1(z)$  are additional potentials that clear the tractions along  $z = x$ . The full-space

689 potentials are given by (e.g. *Bower*, 2009, Section 5.3.12)

$$\Omega_0(z) = \gamma \ln(z - z_0), \quad (\text{C8})$$

$$\omega_0(z) = \bar{\gamma} \ln(z - z_0) - \frac{\gamma \bar{z}_0}{z - z_0}, \quad (\text{C9})$$

690 where

$$\gamma = -\frac{iGbe^{i\beta_0}}{4\pi(1-\nu)}. \quad (\text{C10})$$

691 The additional potentials can be found using a variety of methods (e.g. *Dmowska and Kostrov*,  
692 1973; *Freund and Barnett*, 1976). Here, the additional potentials are computed using the process  
693 of analytic continuation (e.g. Section 3.5 in *England*, 2003), and are given by

$$\Omega_1(z) = -z\overline{\Omega'_0(\bar{z})} - \overline{\omega_0(\bar{z})}, \quad (\text{C11})$$

$$\omega_1(z) = z\overline{\omega'_0(\bar{z})} - \overline{\Omega_0(\bar{z})} + z\overline{\Omega'_0(\bar{z})} + z^2\overline{\Omega''_0(\bar{z})}. \quad (\text{C12})$$

694 Substituting these definitions for  $\Omega_1(z)$  and  $\omega_1(z)$  into equations (C7) along with the results for  
695  $\Omega_0(z)$  and  $\omega_0(z)$ , the potentials for an edge dislocation in a half-space are:

$$\Omega(z) = \gamma \ln \left[ \frac{z - z_0}{z - \bar{z}_0} \right] - \frac{\bar{\gamma}(z - z_0)}{z - \bar{z}_0}, \quad (\text{C13})$$

$$\omega(z) = \bar{\gamma} \ln \left[ \frac{z - z_0}{z - \bar{z}_0} \right] - \frac{\gamma \bar{z}_0}{z - z_0} + \frac{\gamma z}{z - \bar{z}_0} + \frac{\bar{\gamma}(z_0 - \bar{z}_0)z}{(z - \bar{z}_0)^2}. \quad (\text{C14})$$

696 Note that equations (A4) and (A5) in *Rudnicki and Wu* (1995) are the derivatives of equations  
697 (C13) and (C14).

698 The normal  $\sigma_\phi$  and shear  $\sigma_{r\phi}$  stresses on the fault due to a single dislocation are given by the  
699 real and imaginary parts of equation (C5), evaluated using equations (C13) and (C14) at values of  
700  $z$  corresponding to  $\phi = \beta_0$  and  $l \leq \xi \leq l + L$ . For a distribution of dislocations along the length of  
701 the fault, the resultant Burger's vector between neighboring points  $\xi$  and  $\xi + d\xi$  is  $b = (\partial\delta/\partial\xi)d\xi$ ,  
702 where  $\delta(\xi)$  is slip on the fault (*Weertman*, 1996; *Freund and Barnett*, 1976). The stress change

703 functions are found by integrating over the length of the fault, such that

$$T(\xi, \delta) = - \int_l^{l+L} \text{Im} \left\{ \Omega'(z) + \overline{\Omega'(z)} + e^{-2i\beta_0} [z\overline{\Omega''(z)} + \omega'(z)] \right\} \frac{\partial \delta}{\partial s} ds, \quad (\text{C15})$$

$$N(\xi, \delta) = \int_l^{l+L} \text{Re} \left\{ \Omega'(z) + \overline{\Omega'(z)} + e^{-2i\beta_0} [z\overline{\Omega''(z)} + \omega'(z)] \right\} \frac{\partial \delta}{\partial s} ds, \quad (\text{C16})$$

704 where the potentials are evaluated using equations (C13) and (C14) at  $z = \xi e^{i\beta_0}$  and  $z_0 = s e^{i\beta_0}$ .  
 705 Finally, note that it is possible to write the integrands in equations (C15) and (C16) explicitly in  
 706 terms of  $\xi$  and  $\beta_0$ , however the resulting expressions are extremely cumbersome [see for example  
 707 equations (13) in *Freund and Barnett* (1976); equations (3.1) – (3.2) in *Dmowska and Kostrov*  
 708 (1973); or equations (A6) – (A11L) in *Rudnicki and Wu* (1995)]. For numerical computations it is  
 709 most concise to compute the stresses using the individual equations listed above.